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# *Design of Floating Thermal Spacers in an Ultra-Efficient Adiabatic Compressor*

*Lawrie Swinfen-Styles*

*PGR Student, University of Nottingham*

# *Contents*

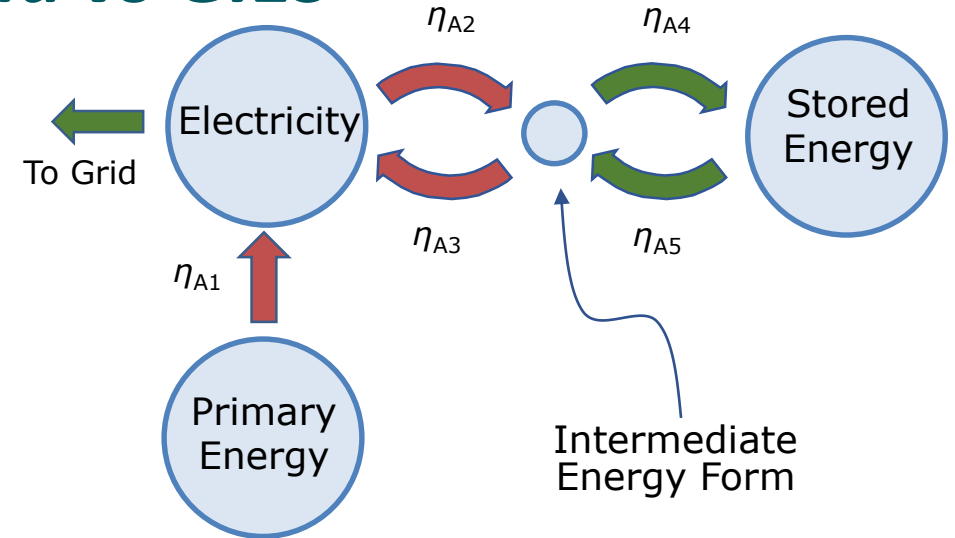
- ***Energy Storage: Mid-Grid vs GIES***
- ***WindTP System Overview***
- **Compressor and Expander Technology**
- **Floating Thermal Spacers and Thermal Gaskets**
- **Forces in Floating Thermal Spacers**

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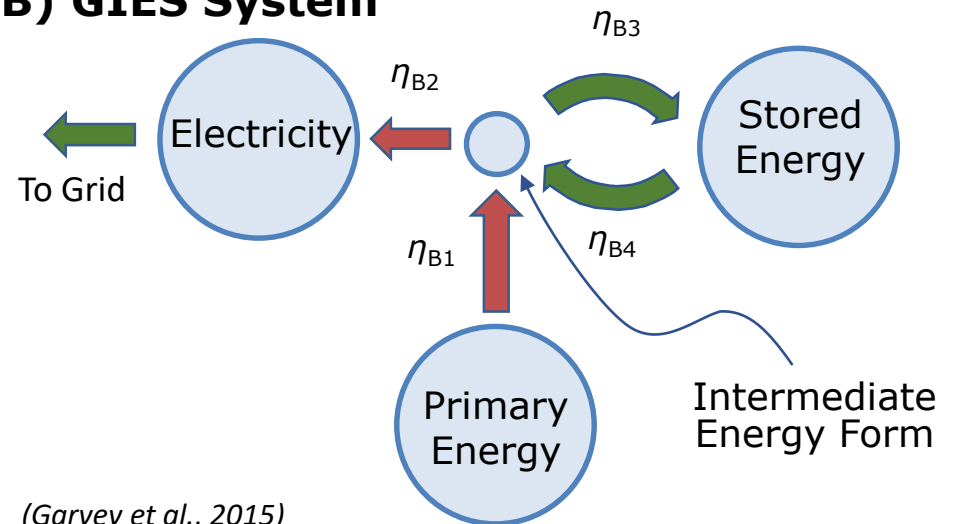
## Energy Storage: Mid-Grid vs GIES

- In a highly-variable future energy mix, large quantities of energy storage will be required; possibly >320 TWh in Europe.
- Most energy storage focus is on mid-grid solutions; electricity is absorbed, stored and returned.
- However, generation-integrated energy storage (GIES) is an alternative set of solutions; energy is stored *before* being converted into electricity.
- Wind turbines with co-located battery banks are not GIES!



**A) Non-GIES System**

**B) GIES System**



(Garvey et al., 2015)

## For more information on GIES:

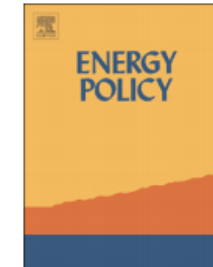
Energy Policy 86 (2015) 544–551



Contents lists available at [ScienceDirect](#)

### Energy Policy

journal homepage: [www.elsevier.com/locate/enpol](http://www.elsevier.com/locate/enpol)



### On generation-integrated energy storage



S.D. Garvey<sup>a,\*</sup>, P.C. Eames<sup>b</sup>, J.H. Wang<sup>c</sup>, A.J. Pimm<sup>a</sup>, M. Waterson<sup>c</sup>, R.S. MacKay<sup>c</sup>,  
M. Giulietti<sup>c</sup>, L.C. Flatley<sup>c</sup>, M. Thomson<sup>b</sup>, J. Barton<sup>b</sup>, D.J. Evans<sup>d</sup>, J. Busby<sup>d</sup>, J.E. Garvey<sup>e</sup>

<sup>a</sup> *University of Nottingham, United Kingdom*

<sup>b</sup> *Loughborough University, United Kingdom*

<sup>c</sup> *University of Warwick, United Kingdom*

<sup>d</sup> *British Geological Survey, United Kingdom*

<sup>e</sup> *University of Leeds, United Kingdom*

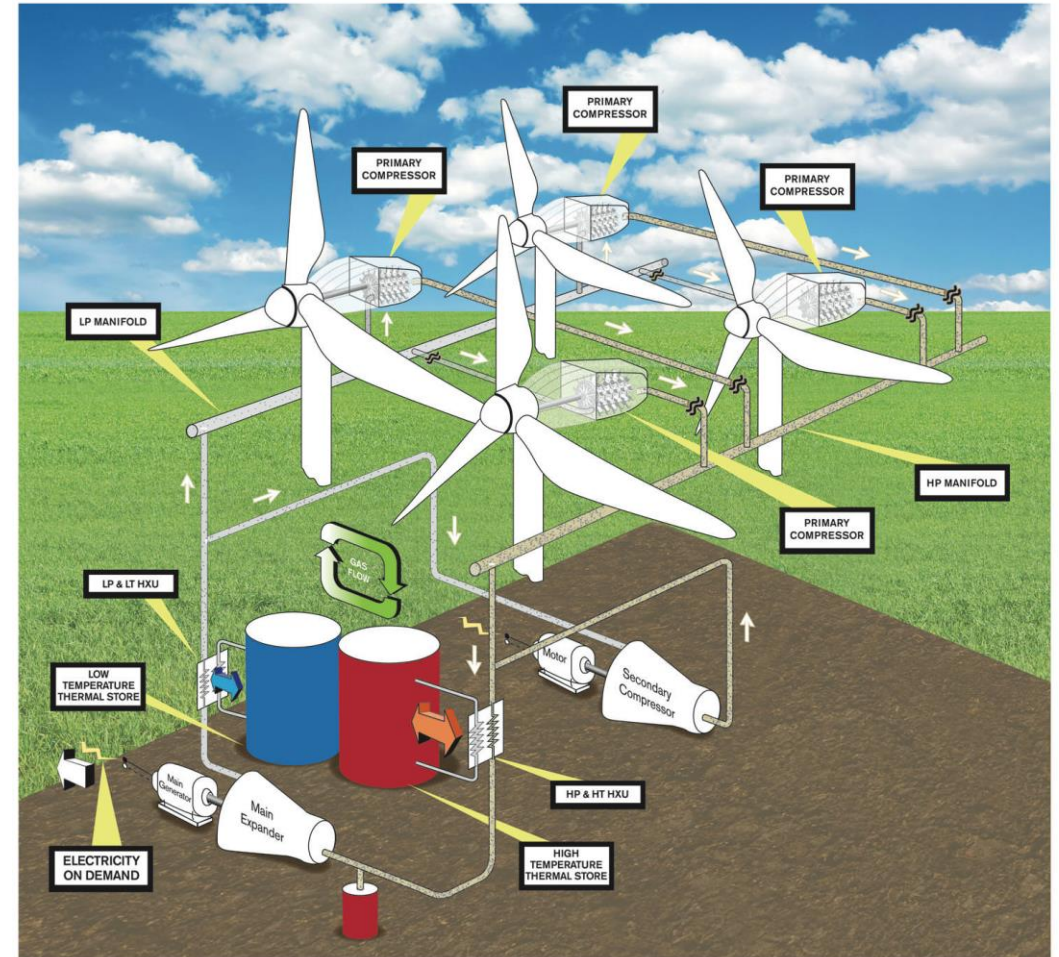
<http://www.sciencedirect.com/science/article/pii/S0301421515300458>

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## WindTP System Overview

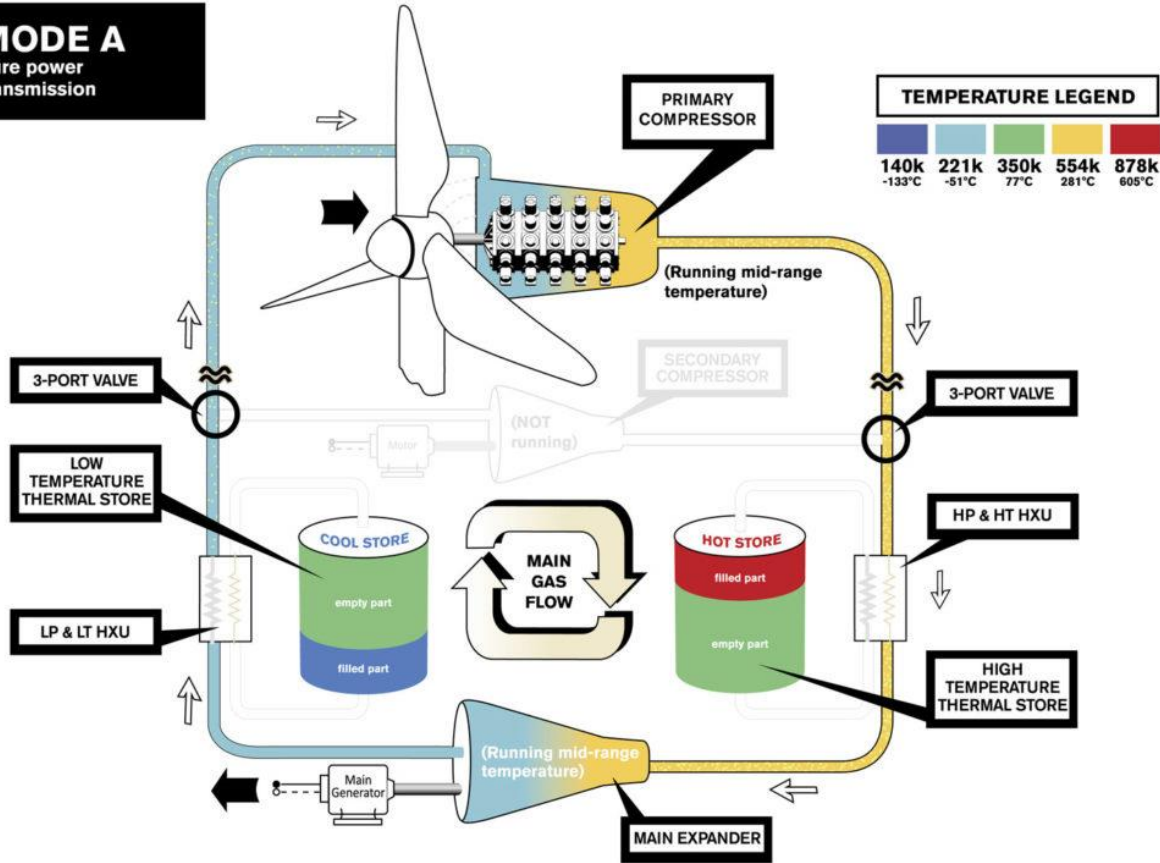
- *WindTP* stands for **Wind-Driven Thermal Pumping**
- Wind turbines directly drive efficient compressors. Gas is compressed adiabatically, increasing its temperature.
- Direct drive removes the need for a gearbox.
- *Thermal* storage, not *compressed air* storage.
- Made from readily available components – with one notable exception!



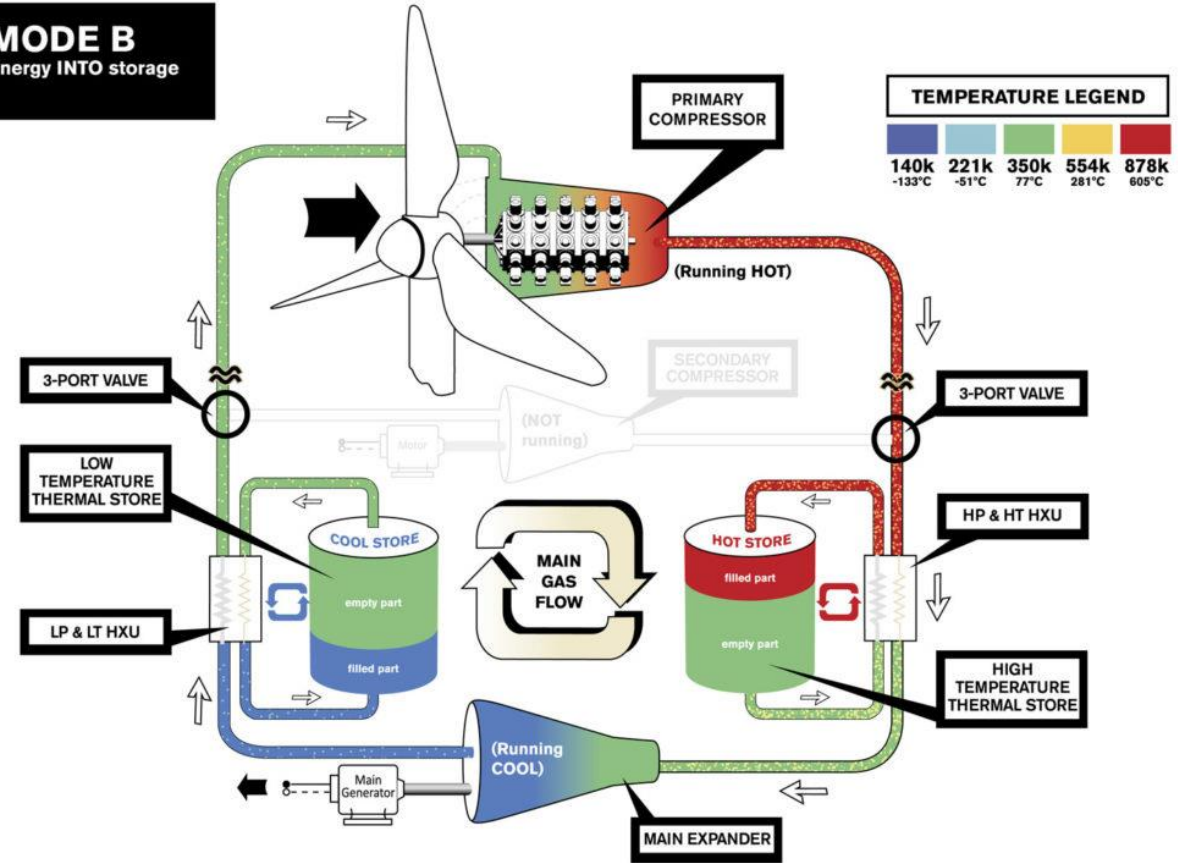
(Source: [www.wind-tp.com](http://www.wind-tp.com))

# WindTP System Overview

## MODE A Pure power transmission



## MODE B Energy INTO storage

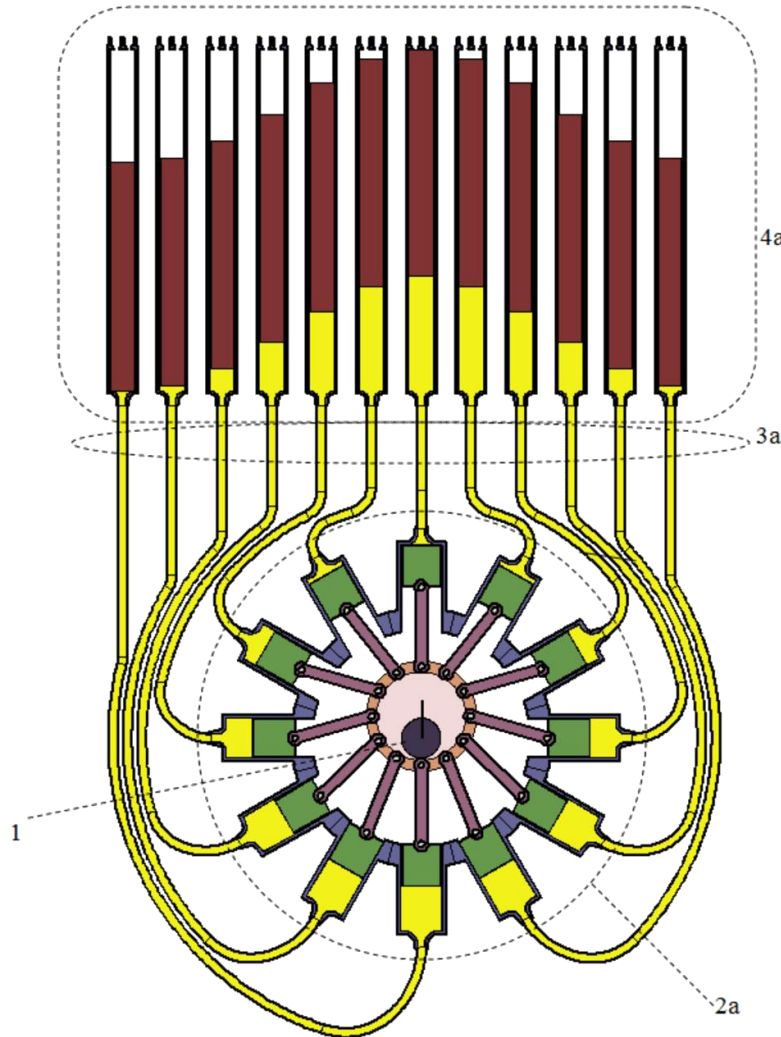




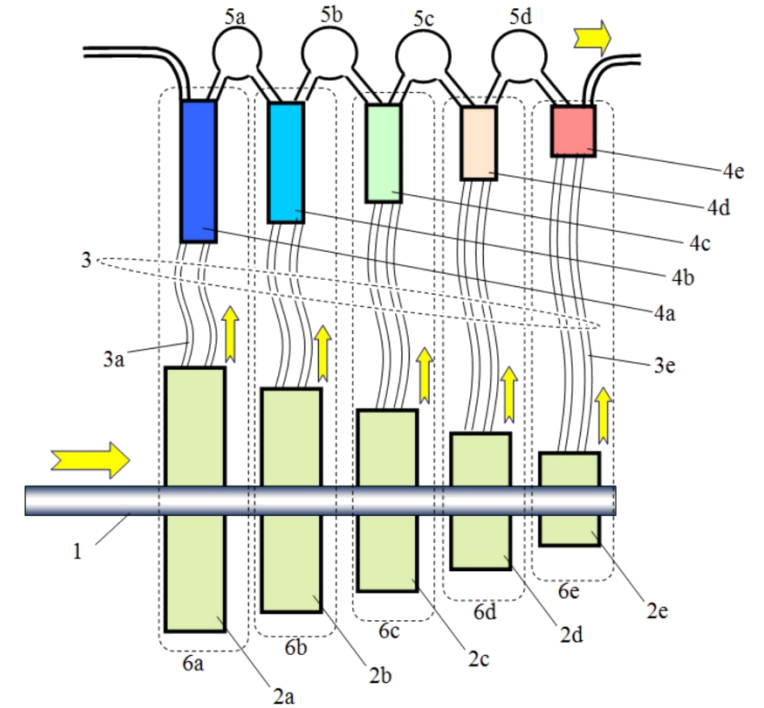
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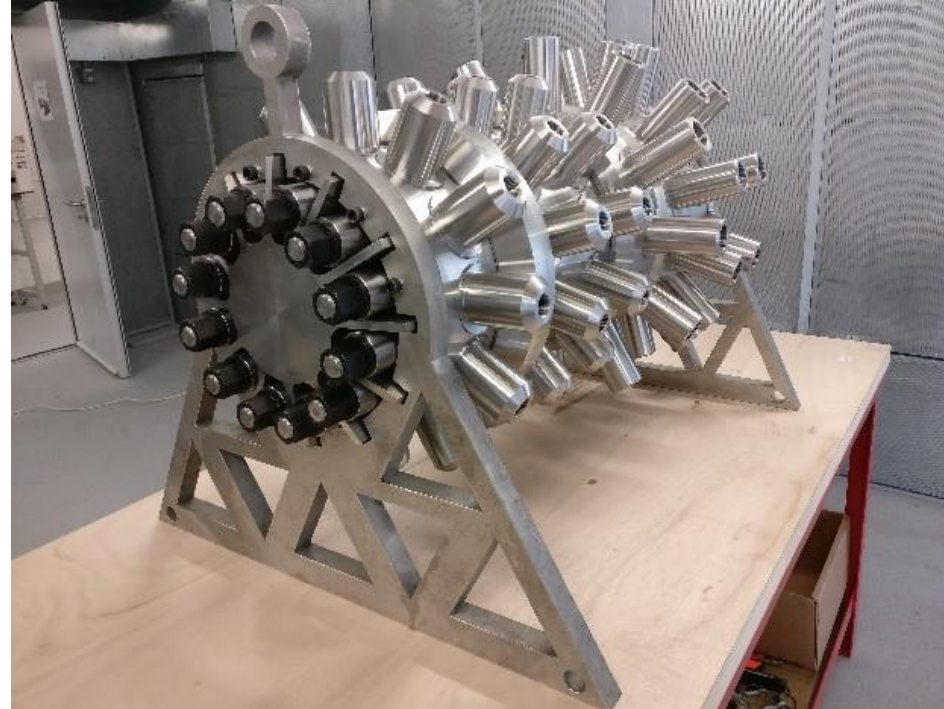
# Compressor and Expander Technology



- Liquid piston adiabatic compressors/expanders.
- High efficiency a priority. Can “afford” to be efficient due to necessarily slow speed.
- Multi-stage to reduce  $\Delta T$  in each stage. Balance of stages to reduce valve losses.
- Split into converter and displacer sections (pictured) to reduce heat transfer.



## *Compressor and Expander Technology*

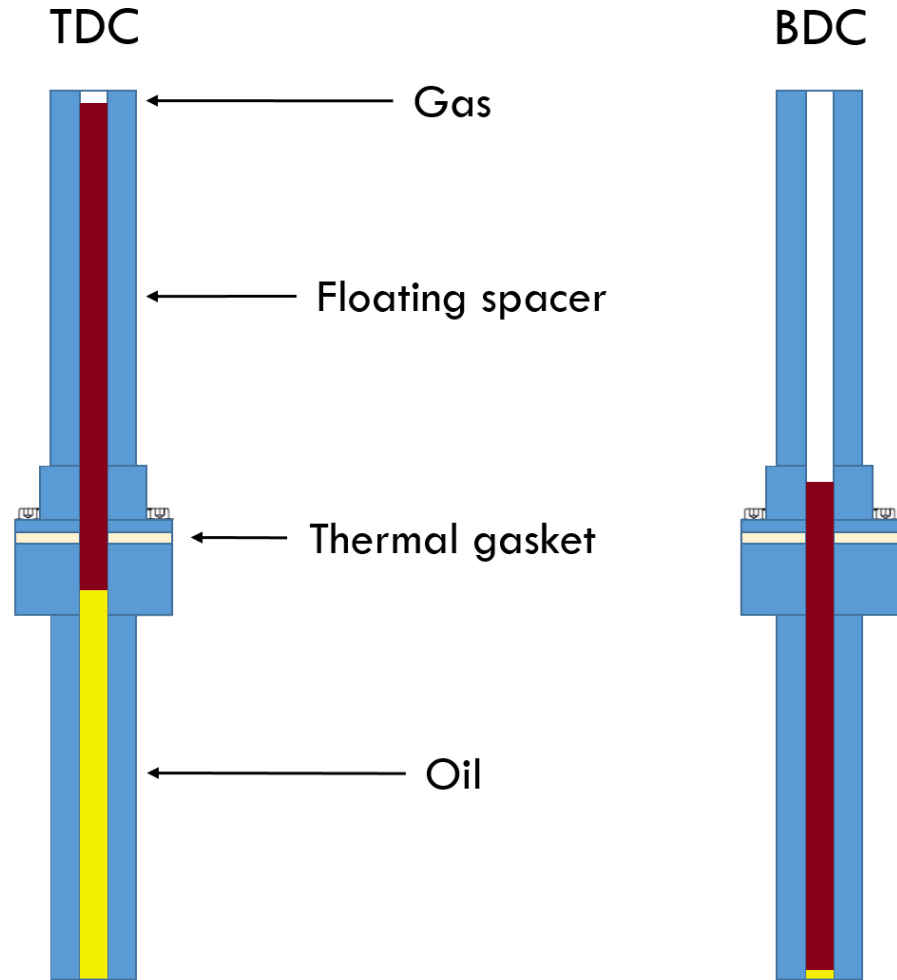
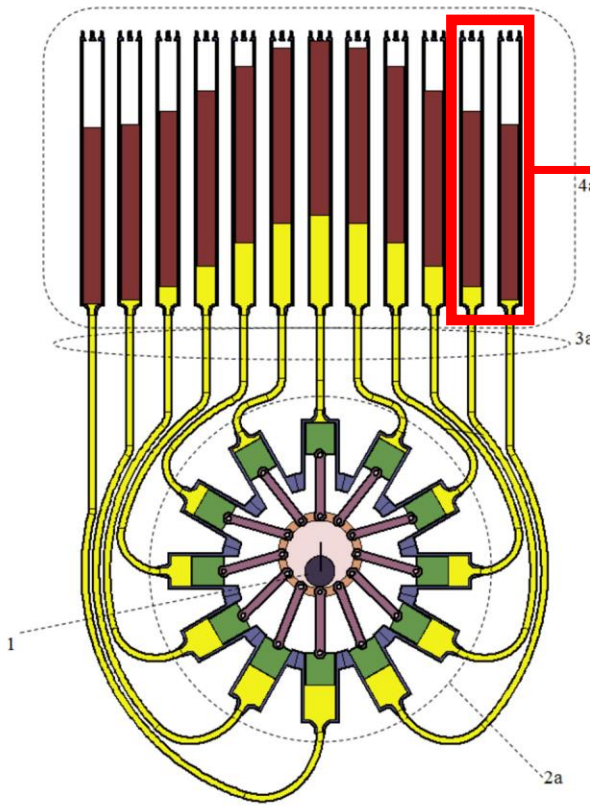


Displacer section for 60kW compressor/expander prototype.

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# Floating Spacers and Thermal Gaskets



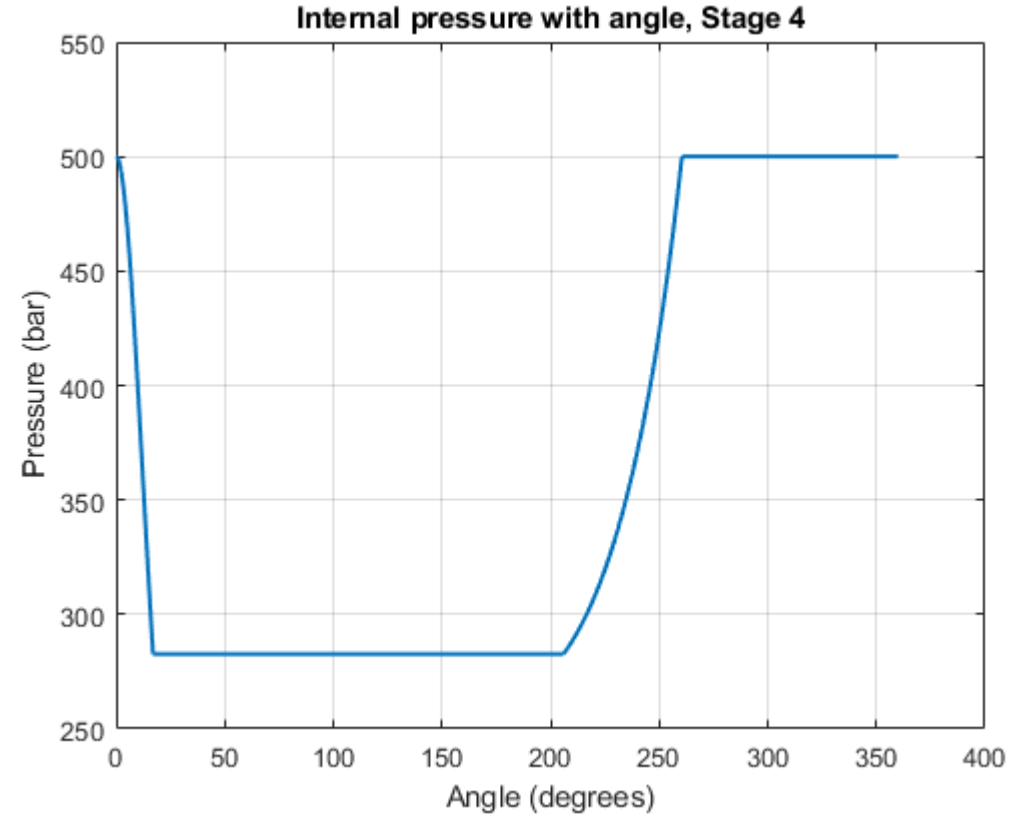
With the floating spacers and thermal gaskets working in tandem, there is no thermal path down the sides of the pipes.

## *Floating Spacer Requirements*

Intended to thermally separate the extreme-temperature gas from the oil.

Material requirements:

- Extremes of temperature ( $-133^{\circ}\text{C}$  and  $+600^{\circ}\text{C}$ ).
- High – and fluctuating – pressures (e.g. 282 bar to 500 bar in the HP end).
- Very low thermal conductivity.
- High stiffness. Nothing spongy!
- LIGHT – preferably lifted entirely by the oil.



## *A Suitable Material?*

Fused silica glass:

- 1100°C upper working temperature.
- >1000 MPa compressive strength.
- 1.38 W/m·K thermal conductivity.
- High stiffness.
- 2200kg/m<sup>3</sup>. Still not light enough...?



## *Evacuated Tubes and Springs*

By evacuating the tubes, we improve several properties:

- Lower overall thermal conductivity.
- All material in the spacer is constantly in compression – good for glass!
- Bulk density of the spacer is reduced significantly – much closer to the density of the oil (around  $700 \text{ kg/m}^3$ ).



The mass not supported by the buoyancy force can then be held with (very) light springs. A second compressor design could increase the space available for oil to rise up the thermal spacer, increasing the potential buoyancy force.

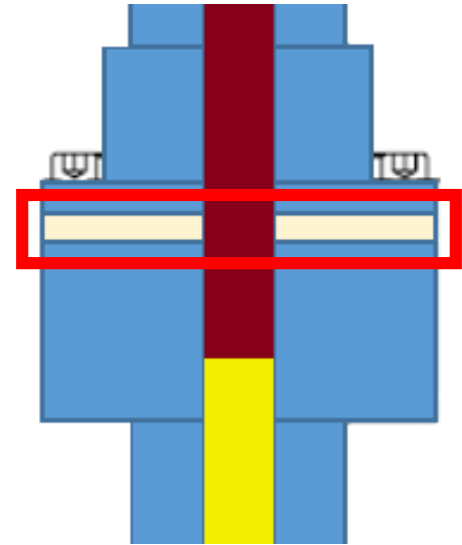


## *Thermal Gasket Requirements*

Used to remove the potential thermal path down the walls of the pipes.

Material requirements:

- Low thermal conductivity.
- Able to withstand high temperature gradient over a very short distance.
- Able to seal against pressures up to 500 bar.
- Similar coefficient of thermal expansion to stainless steel.



## *A Suitable Material?*

Yttria-stabilised zirconia:

- 2.5 W/m·K thermal conductivity.
- >2000 MPa compressive strength.
- Can be ground optically flat for sealing – used in quarter-turn taps!
- $7 \times 10^{-6}$  to  $11 \times 10^{-6}$ /K coefficient of thermal expansion (compared with  $15 \times 10^{-6}$ /K for stainless steel).



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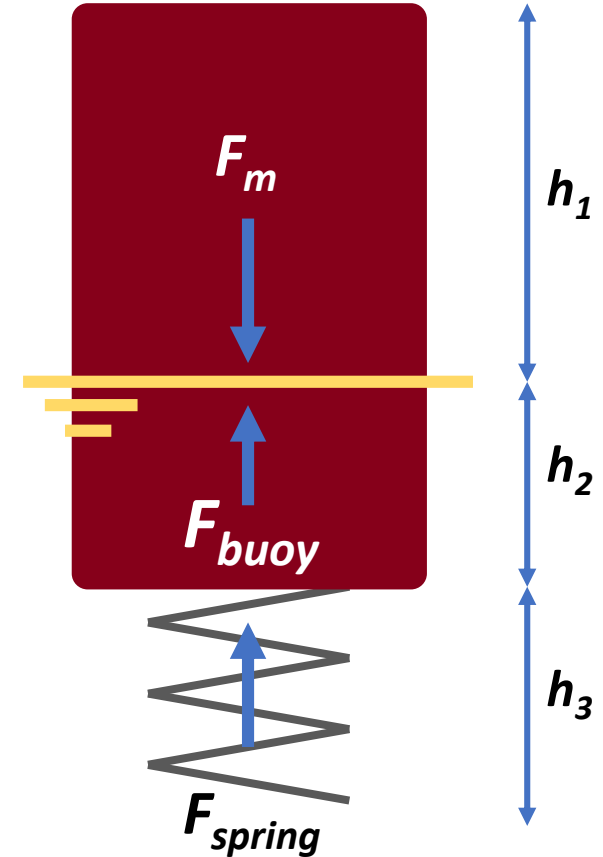
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## Forces in Floating Thermal Spacers

$F_m$  – force due to floating spacer mass. Constant.

$F_{buoy}$  – buoyancy force in oil. Dependent on  $h_2$  **and on gas density** (increased gas density increases hydrostatic pressure at base of spacer). Force **increases** during gas compression from BDC to TDC.

$F_{spring}$  – spring force. Dependent on  $h_3$  and spring parameters. Force **decreases** during gas compression from BDC to TDC (due to reduced compression of spring).





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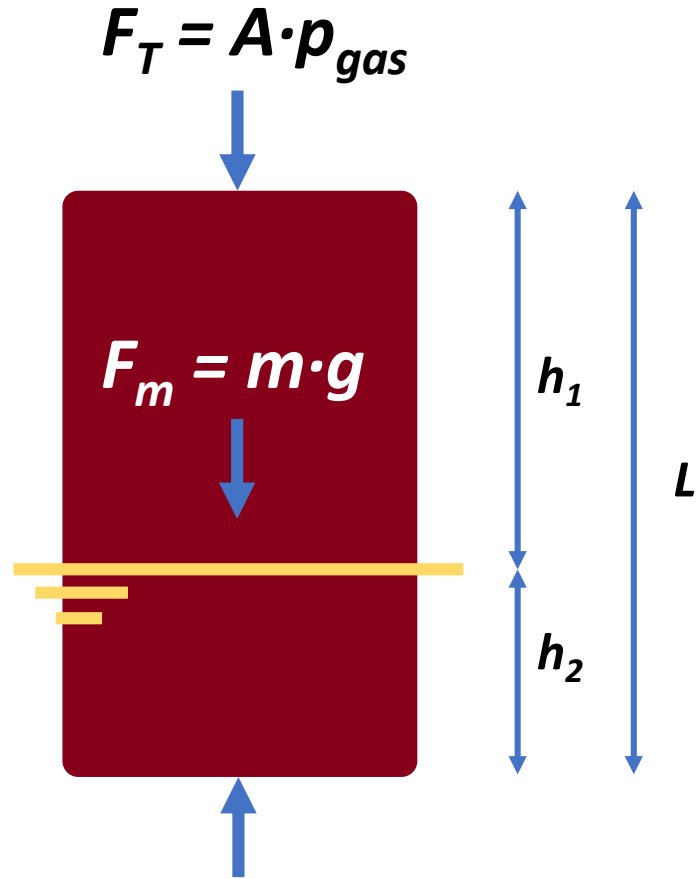


# Thank you for listening.

If you would like to contact me:  
[lawrie.swinfen-styles@nottingham.ac.uk](mailto:lawrie.swinfen-styles@nottingham.ac.uk)

For more information on *WindTP*:  
[www.wind-tp.com](http://www.wind-tp.com)

## Extra Slide - Forces in Floating Thermal Spacers



$$mg + Ap_{gas} = A(p_{gas} + \rho_{gas}gh_1 + \rho_{oil}gh_2)$$

$$m/A = \rho_{gas}h_1 + \rho_{oil}h_2$$

$$L = h_1 + h_2$$

$$m/A = \rho_{gas}L - \rho_{gas}h_2 + \rho_{oil}h_2$$

$$h_2 = [(m/A) - \rho_{gas}L] / [\rho_{oil} - \rho_{gas}]$$

$$F_B = A(p_{gas} + \rho_{gas} \cdot g \cdot h_1 + \rho_{oil} \cdot g \cdot h_2)$$